

HOMEWORK #6

Some properties of the $k_t - \epsilon$ turbulence model

We consider some properties of the $k_t - \epsilon$ turbulence model, widely used in industry for solving the averaged Navier-Stokes equations. The governing equations are first recalled. For an incompressible flow, the conservation of mass and of momentum are given by

$$\frac{\partial \bar{U}_i}{\partial x_i} = 0 \quad (1)$$

$$\frac{\partial(\rho \bar{U}_i)}{\partial t} + \frac{\partial(\rho \bar{U}_i \bar{U}_j)}{\partial x_j} = -\frac{\partial}{\partial x_i} \left(\bar{P} + \frac{2}{3} \rho k_t \right) + \frac{\partial}{\partial x_j} \left[2(\mu + \mu_t) \bar{S}_{ij} \right] \quad (2)$$

where the mean deformation tensor reads $\bar{S}_{ij} = (\partial \bar{U}_i / \partial x_j + \partial \bar{U}_j / \partial x_i) / 2$. The Reynolds tensor is closed thanks to the introduction of a turbulent viscosity μ_t ,

$$-\overline{\rho u'_i u'_j} = 2\mu_t \bar{S}_{ij} - \frac{2}{3} \rho k_t \delta_{ij} \quad (3)$$

and this flow dependent viscosity is calculated from the turbulent kinetic energy k_t and its dissipation rate ϵ as follows,

$$\mu_t = \rho C_\mu \frac{k_t^2}{\epsilon} \quad (4)$$

Two additional transport equations are solved to compute k_t and $\epsilon \simeq \epsilon^h$ (the superscript h will be disregarded to simplify notations) for a high Reynolds number flow,

$$\begin{cases} \frac{\partial(\rho k_t)}{\partial t} + \frac{\partial(\rho k_t \bar{U}_i)}{\partial x_i} = \frac{\partial}{\partial x_j} \left[\left(\mu + \frac{\mu_t}{\sigma_k} \right) \frac{\partial k_t}{\partial x_j} \right] + \mathcal{P} - \rho \epsilon \\ \frac{\partial(\rho \epsilon)}{\partial t} + \frac{\partial(\rho \epsilon \bar{U}_i)}{\partial x_i} = \frac{\partial}{\partial x_j} \left[\left(\mu + \frac{\mu_t}{\sigma_\epsilon} \right) \frac{\partial \epsilon}{\partial x_j} \right] + \frac{\epsilon}{k_t} (C_{\epsilon 1} \mathcal{P} - C_{\epsilon 2} \rho \epsilon) \end{cases} \quad (5)$$

where \mathcal{P} is the production term of the turbulent kinetic energy k_t

$$\mathcal{P} = -\overline{\rho u'_i u'_j} \frac{\partial \bar{U}_i}{\partial x_j} \quad (6)$$

In this model, five calibration constants must be determined. For the standard formulation, they are given by

$$C_\mu = 0.09 \quad C_{\epsilon 1} = 1.44 \quad C_{\epsilon 2} = 1.92 \quad \sigma_k = 1.0 \quad \sigma_\epsilon = 1.3 \quad (7)$$

1. Production term

1. Show that the production term can be expressed as $\mathcal{P} = 2\mu_t \bar{S}_{ij}^2$
2. By considering the function $f(\lambda) = \overline{(u'_i + \lambda u'_j)^2}$, show that the Reynolds tensor must satisfy Schwarz's inequality, that is

$$\overline{u'_i u'_j}^2 \leq \overline{u'_i{}^2} \overline{u'_j{}^2}$$

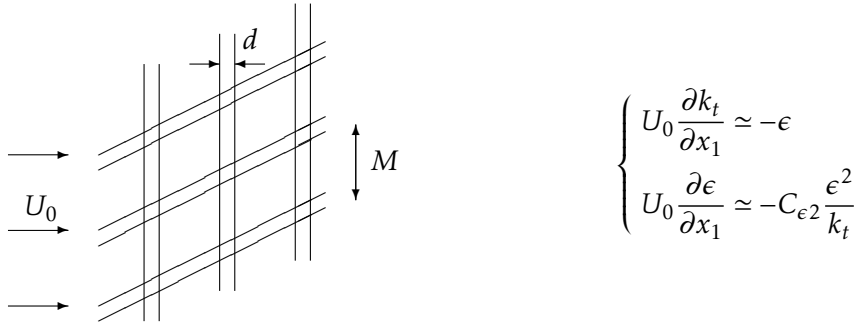
3. For a two-dimensional mean shear flow, $\overline{U}_1 = \overline{U}_1(x_2)$ and $\overline{U}_2 = \overline{U}_3 = 0$, show that the $k_t - \epsilon$ model imposes the following relationship

$$9C_\mu^2 \overline{S}_{12}^2 \leq \frac{\epsilon^2}{k_t^2} \sim \omega_t^2$$

How can one interpret ω_t ?

2. Determination of the C_{ϵ_2} constant

For grid-generated homogeneous turbulence, the production term $\mathcal{P} \equiv 0$ is zero and turbulence is simply convected at a constant velocity $\overline{U}_1 = U_0$ and decays behind the grid. The two transport equations of the system (5) are reduced to



4. Show that the solution to this system of ordinary differential equations can be recast in the following form,

$$k_t = k_{t0} \left[1 + (C_{\epsilon_2} - 1) \frac{\epsilon_0 x_1}{k_0 U_0} \right]^{-\frac{1}{C_{\epsilon_2} - 1}} \quad \text{and} \quad \frac{\epsilon}{\epsilon_0} = \left(\frac{k_t}{k_{t0}} \right)^{C_{\epsilon_2}}$$

where $k_t = k_{t0}$ and $\epsilon = \epsilon_0$ at $x_1 = 0$.

5. Experimentally, refer to the measurements by Comte-Bellot & Corrsin (1966) among others, the following decaying law is observed $(k/k_0) \sim (t/t_0)^{-n}$ with $n \simeq 1.3$, in a frame convected at the velocity vitesse U_0 , consequently $t = x_1/U_0$. Deduce a numerical estimate of the constant C_{ϵ_2} .
6. Provide the evolution of the length scale $L = k_t^{3/2}/\epsilon$? and give an interpretation of this scale.

3. Calibration of the C_μ and C_{ϵ_1} constants

For turbulent shear flow, in particular in the log-law of a turbulent boundary layer where $\mathcal{P} \simeq \rho\epsilon$, refer to Bradshaw *et al.* (1967), one also observes that

$$-\frac{\overline{u'_1 u'_2}}{k_t} \simeq 0.30$$

To determine the value of C_μ , consider the turbulent boundary layer established in a plane channel flow, with $\overline{U}_1 = \overline{U}_1(x_2)$, $\overline{U}_2 = \overline{U}_3 = 0$.

6. By examining the balance $\mathcal{P} \simeq \rho\epsilon$, show that

$$-\frac{\overline{u'_1 u'_2}}{k_t} = C_\mu^{1/2}$$

and provide a numerical estimate of C_μ .

7. Provide the expressions of $d\bar{U}_1/dx_2$ and ϵ inside the log-law of a turbulent boundary layer. Show also that $\nu_t \simeq \kappa u_\tau x_2$ and that $k_t \simeq u_\tau^2 / \sqrt{C_\mu}$.

8. By noting that the two differential equations of the system (5) simplify in the considered flow to

$$0 = \mathcal{P} - \rho\epsilon \quad \text{and} \quad 0 = \frac{d}{dx_2} \left[\left(\frac{\mu_t}{\sigma_\epsilon} \right) \frac{d\epsilon}{dx_2} \right] + \frac{\epsilon}{k_t} (C_{\epsilon 1} \mathcal{P} - C_{\epsilon 2} \rho\epsilon)$$

show that the compatibility relation must be satisfied,

$$\sigma_\epsilon C_\mu^{1/2} (C_{\epsilon 2} - C_{\epsilon 1}) = \kappa^2$$

where κ is the von Kármán constant. Comment this result.